Degraded Dampers

A Case study in Engine Torsional Vibration



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Overview

- The problem
- Investigation
 - Fractography
 - Engine instrumentation
- Analysis and discussion
- Conclusions



The problem

- Accumulated 6 in-flight failures of engine driven fuel pump (EDFP) couplings
- Resulting 2 Mayday and 4 Pan calls
- 1 cracked coupling found during inspection
- Failures at substantially less than normal life







	Probability of failure
Lycoming recommended TBO, SI1009AS, 1400 hours	1%
Lycoming approved TBO, 1800 hours	5%
Extended TBO, 2000 hours	10%

Engine configuration



Component Investigation

- Check component suitability
 - Fractography
 - Materials analysis
 - Finite element analysis





- All failures located at waisted portion of shaft
- Exhibited faceting at 90° and ±45°, indicative of reversed torsional and bending fatigue loads
- Coupling splines and drive gear burnished on both flanks indicative of oscillatory torsional loading
- Analyses by 3 laboratories show no material or component deficiencies







Fractography / material analysis, 2 of 2





• Resonance is caused by exciting a structure at its natural frequency



•Engine/pump combination unique to RNZAF fleet

Finite Element Analysis to determine coupling natural frequency
17,692 Hz or approximately 1,000,000 RPM!
Highest stress location not at failure location
Therefore coupling resonance improbable



- EDFP manufacturer
 - Component failure due to torsional fatigue
- Engine manufacturer
 - Component failure due to component defect, engine within limits



Instrumented engine tests

- Assess the 3 failure aircraft for differences from un-failed fleet to identify potential sources of torsional loading
 - Engine run up in increments from idle to full power
 - Parameters recorded
 - EDFP vibration (3-axis linear)
 - Fuel pressure at EDFP inlet, outlet and auxiliary pump outlet
 - Fuel temperature (manually recorded)





Analysis and discussion – fuel pressure?



- EDFP load reversal indicated
- Solve failures by increasing EDFP relief valve pressure setting? No

Analysis and discussion – vibration?



Vibration on engine with no failures, NZ1990 lateral vibration



•Fleet managed by identifying susceptible aircraft, however root cause still unknown

- No detectable EDFP coupling resonance
- No significant temperature fluctuations
- No detectable cavitation or vapour lock
- Amplitude modulation present in EDFP vibration on failed EDFP
- Fuel pipe sizing, fuel pressure setting, propeller balance, propeller type, magneto condition, engine mount stiffness and governor condition were found not to change measured parameters
- Root cause undetermined...

- Addition of torsional vibration measurement to examine EDFP coupling motion
- Installation of a modified EDFP
 - Omron optical rotary encoder, attached to pump rotor and installed in a modified pump
 - EDFP run dry and fuel bypassed. Fuel supplied by backup electrically driven pump
 - Rotary encoder outputs 100 pulses per revolution (1 pulse per 3.6 degrees)







Torsional vibration on an engine with failures, NZ1988 EDFP speed

Torsional vibration on an engine with no failures, NZ1996 EDFP speed





Frequency (Hz)

• Results indicate probable degraded crankshaft torsional dampers

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Crankshaft-EDFP coupling interaction



 Torsional vibration from the engine crankshaft is transmitted through the accessory gears to the EDFP

Engine strip investigation – 1 of 3

- Engine selected which had experienced 4 EDFP coupling failures
- Left hand magneto drive (torsional) damper had failed



Engine strip investigation – 2 of 3

- Excessive wear on accessory drive train
- Gear backlash results are not repeatable







• Crankshaft and counterweight bushes were worn beyond serviceable limits. One bush, 5 times over the serviceable limits







Lycoming AEIO-540-L1B5, EDFP coupling 5th engine order torsional vibration

• Significant decrease in 5th engine order torsional vibration, across the entire engine operating range



Lycoming AEO-540-L1B5, EDFP coupling 6th engine order torsional vibration

- Significant decrease in 6th engine order torsional vibration, across the entire engine operating range
- Confirmed the assumption that crankshaft damper condition is the probable cause of EDFP coupling failure



- Textron-Lycoming SB245D (1987), Dynamic Counterweight System Detuning
 - Warns against rapid throttle operation, high engine speed and low manifold pressure
- Aircraft usage profile
 - Training
 - Aerobatics
- No evidence that specific aircraft have been preferentially operated for aerobatics
 - Why do some aircraft have worn counterweight dampers while others do not?
- Overhauls history showed that counterweight bush replacement varied across the fleet of aircraft

OEM guidance

A Textron Company		MANDATORY		ing	SERVICE	
Williamsport, PA 17701 U.S.A. Tel. 570-323-6181 Fax. 570-327-7101 www.lycoming.textron.com		SERVICE BULLETIN	Williamsport Plant Textron Lycoming/Subsidiary of Textro 652 Oliver Street Williamsport, PA 17701 U.S.A.	on Inc.	INSTRUCTION	
DATE:	May 25, 2006	Service Bulletin No. 240T (Supersedes Service Bulletin No. 240S)				
SUBJECT:	Mandatory Par Normal Mainte	ts Replacement at Normal Overhaul and During Repair or nance	DATE:	April 11, 1988 (Supersedes Service Instruction No. 1143B) Engineering Aspects are FAA Approved		
MODELS AFFECTED:	All Lycoming r	eciprocating aircraft engines.				
TIME OF COMPLIANCE:	As specified be	low.	SUBJECT:	SUBJECT: PART I · Counterweight Bushing Tooling Update.		
AT OVERHAUL:				PART II - Cour	nterweight Bushing Replacement.	
During overhaul of any Lycoming reciprocating engine. It is mandatory that the following parts be replaced regardless of their apparent condition.		MODELS AFFECTED:	All Textron Lycoming piston aircraft engines with dynamic counterweights employing 3/4 inch I.D. bushings.			
			TIME OF COMPLIANCE:	During engine of	verhaul.	
Service Instructions No. 1142 and 1143 for instructions.)		Dynamic counterweights are installed on piston engine crankshafts to eliminate vibrations that are caused by torsional frequency changes that occur at different engine speeds and operating conditions. The counterweight is mounted on the crankshaft				

 Mandatory SB reference a condition based SI

> Counterweight Bushing Inspection - Wear in the counterweight bushings is usually evident as out-of-round on the inside diameter. Check each bushing with the P/N ST-73 bore gage. The diameter should be between 0.7485 and 0.7505 inch and the out-of-round should not exceed 0.0005 inch. The P/N ST-73 gage can be set with a micrometer. If the diameter of any bushing is oversize or out-ofround all of the bushings in the counterweight must be replaced.

> with two steel rollers that allow the counterweight to move as required to maintain crankshaft balance. Both the counterweight supporting lug of the crankshaft and the counterweight contain hardened

> steel bushings that are ground to a very smooth and annular finish. If any of these bushings become

> damaged or worn out of round, the counterweight will become ineffective and cause vibrations that will lead to severe engine damage or failure.



- EDFP coupling failure was caused by operation with degraded crankshaft counterweight dampers
- Counterweight degradation can lead to failure of major engine components
- Crankshaft torsional vibration is not easily measured. Gear backlash an ineffective measure
- Accessory gear condition is a measure of accumulated damage
- Flexibility regarding application of mandatory SB240T and SI1143B can lead to default acceptance of airworthiness risk
- SB245D (1987) is pilot centric however pilots may not have visibility of SB. Usage monitoring would be beneficial
- SB245D Dynamic Counterweight System Detuning still appropriate, should be considered when setting TBO.

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